# EFFECT OF RADIATION, DISSIPATION AND THERMO-DIFFUSION ON CONVECTIVE HEAT AND MASS TRANSFER FLOW OF A CHEMICALLY REACTING FLUID IN A NON-UNIFORMLY HEATED VERTICAL CHANNEL

# <sup>1</sup>R. Mabuni<sup>\*</sup>, <sup>1</sup>U. Rajeswara Rao, and <sup>2</sup>D. R. V. Prasada Rao

<sup>1</sup>Department of Mathematics, Sri Krishnadevaraya University, Ananthapuramu-515003, (A.P.), India.

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# ABSTRACT

In this Paper, we discuss the effect of radiation on mixed convective heat and mass transfer flow of a viscous, electrically conducting, and incompressible fluid through a porous medium in a vertical channel bounded by flat walls. A non-uniform temperature is imposed on the walls and the concentration on these walls is taken to be constant. The viscous dissipation is taken in to account in the energy equation. Assuming the slope of the boundary temperature to be small. We solve the governing momentum, energy and diffusion equations by a perturbation technique. The velocity, the temperature, the concentration, the shear stress and the rate of heat transfer have been analysed for different variations of the governing parameters. The dissipative effects on the flow, heat and mass transfer are clearly brought out.

Keywords: Radiation, Dissipation, Thermo-Diffusion, Heat and Mass Transfer, Chemical Reaction, Vertical Channel.

# **1. INTRODUCTION**

Flows which arise due to the interaction of the gravitational force and density differences caused by the simultaneous diffusion of thermal energy and chemical species, have many applications in geophysics and engineering. Such thermal and mass diffusion plays a dominant role in a number of technological and engineering systems. For example, in controlling surface temperature by evaporation, cooling controlling polymerization reaction products by injecting suitable reactants along the porous wall of the reactor, distillation of volatile components from a mixture with non – volatiles, are a few technological process in which mass transfer accompanies the transfer of heat. Likewise such combined heat and mass transfer occurs in natural environmental vapourisation of mist and fog, evaporation in terrestrial bodies of water viz. the oceans, rivers and the ponds, drying of food grains, simultaneous diffusion of metabolic heat and perspiration to control body temperature. Obviously the understanding of this transport process is desirable in order to effectively control the overall transport characteristics. The combined effect of thermal and mass diffusion in channel flows has been studied in the recent times by a few authors (17, 24,31,41,47, 46, 52, 54).

Flow through porous media is very prevalent in nature, and therefore the study of flow through porous medium has become of principal interest in many scientific and engineering applications. In the theory of flow through a porous medium, the role of momentum equation or force balance is occupied by the numerous experimental observations summarized mathematically as the Darcy's law. It is observed that the Darcy's law is applicable as long as the Reynolds number based on average grain (pore) diameter does not exceed a value between 1 and 10. But in general the speed of specific discharge increases, the convective forces get developed and the internal stress generated in the fluid due to its viscous nature produces distortions in the velocity field. Also in the case of highly porous media such as fiber glass, papus of dendilion etc., the viscous stress at the surface is able to penetrate into medium and produce gradient. Thus between the specific discharge and hydraulic gradient is inadequate in describing high speed flows or flows near surfaces which may be either permeable or not. Hence consideration for non-Darcian description for the viscous flow through porous media is warranted. Saffman (49) employing statistical method derived a general governing equation for the flow in a porous medium which takes into account the viscous stress. Later another modifications has been suggested by Brinkman (9).

$$0 = -\nabla P - \frac{\mu}{k}\overline{v} + \mu\nabla^2\overline{v}$$

*Corresponding author:* <sup>1</sup>*R. Mabuni*\*

in which  $\mu \nabla^2 \overline{v}$  is intended to account for the distortions of the velocity profiles near the boundary. The same equation was derived analytically by Tam (56) to describe the viscous flow at low Reynolds number past a swam of small particles.

This problem of combined buoyancy driven thermal and mass diffusion has been studied in parallel plate geometries by a few authors in the recent times, notably Gebhart (21), Lai(33,33a), Chen and Mout Sogloy (14), Poulikakos (44), Trevisan and Bejan (57), Mehta and Nanda Kumar(40) and Angiras *et al*(1).

In the risk assessment of nuclear power plants, the possibility and the consequences of a melt down of the reactor core are usually considered. During the course of such an accident molten fuel and coolant may interact. Violent thermal reaction can dispose the molten fuel into fine particles. These small particles quickly solidify in the coolant and settle on internal structures of the reactor pressure vessel forming a saturated porous bed. The question arises, under what conditions the nuclear decay heat can be removed from the particle bed to the ambient coolant by natural convection. Thus the problem of natural convection in saturated porous layers. This analysis of heat transfer in a viscous heat generating fluid also important in engineering processes pertain to flow in which a fluid supports an exothermal chemical or nuclear reaction or problems concerned with dissociating fluids(34,36). The Volumetric heat generation has been assumed to be constant (2,6,7,7a,8,15,42,43) or a function of space variable (12,14,25, 26,29,37). For example a hypothetical core-disruptive accident in a liquid metal fast breeder reactor (LMFBR) could result in the setting of fragmented fuel debris as horizontal surfaces below the core. The porous debris could be saturated sodium coolant and heat generation will result from the radioactive decay of the fuel particulate(20) The heat losses from the geothermal system in some cases can be treated as if the heat comes from the heat generating sources (27). Keeping this in view, porous medium with internal heat source have been discussed by several authors (11, 20, 27, 28, 43).

In the above mentioned investigations the bounding walls are maintained at constant temperature. However, there are a few physical situations which warrant the boundary temperature to be maintained non-uniform. It is evident that in forced or free convection flow in a channel (pipe) a secondary flow can be created either by corrugating the boundaries or by maintaining non-uniform wall temperature such a secondary flow may be of interest in a few technological process. For example in drawing optical glass fibres of extremely low loss and wide bandwidth, the process of modified chemical vapour deposition (MCVD) [32, 50] has been suggested in recent times. Performs from which these fibres are drawn are made by passing a gaseous mixture into a fused–silica tube which is heated locally by an oxy-hydrogen flame. Particulates of Sio<sub>2</sub>- Geo<sub>2</sub> composition are formed from the mixture and collect on the interior of the tube. Subsequently these are fined to form a victreous deposit as the flame traversed along the tube. The deposition is carried out in the radial direction as the flame traversed along the tube. The deposition is carried out in the radial direction through the secondary flow created due to non-uniform. Ravindranath *et al* (58) have studied the combined effect of convective heat and mass transfer on hydromagnetic electrically conducting viscous incompressible fluid through a porous medium in a vertical channel bounded by flat walls which are maintained at non-uniform temperatures.

All the above mentioned studies are based on the hypothesis that the effect of dissipation is neglected. This is possible in case of ordinary fluid flow like air and water under gravitational force. But this effect is expected to be relevant for fluids with high values of the dynamic viscous flows. Moreover Gehart(22), Gebhart and Mollen dorf(23) have shown that that viscous dissipation heat in the natural convective flow is important when the flow field is of extreme size or at extremely low temperature or in high gravitational filed. On the other hand Barletta(3) has pointed out that relevant effect of viscous dissipation on the temperature profiles and on the Nusselt numbers may occur in the fully developed forced convection in tubes. In view of this several authors notably, Soudalgekar and Pop(53) Raptis et al (48), Ramana Murthy and Soundelgekar *et.al* (51), Barletta (3,4), Sreevani(55). El-hakeing (18), Bulent Yesilata(10). Rossidi schio(48a) and Israel et al (30) have studied the effect of viscous dissipation on natural convection has been studied for some different cases including the natural convection from horizontal cylinder. The natural convection from horizontal cylinder embedded in a porous media has been studied by Fand and Brucker(19). They reported that the viscous dissipation may not be neglected in all cases of natural convection from horizontal cylinders and further, that the inclusion of a viscous dissipation term in porous medium may lead to more accurate correlation equations.

The effect of viscous dissipation has been studied by Nakayama and Pop (38) for steady free convection boundary layer over a non-isothermal bodies of arbitrary shape embedded in porous media. They used integral method to show that the viscous dissipation results in lowering the level of the heat transfer rate from the body. Costa (16) has analysed a natural convection in enclosures with viscous dissipation. Recently Jambal *et.al* have discussed the effects of viscous dissipation and fluid axial heat conduction heat transfer for non-Newtonian fluids in ducts with uniform wall temperature. Prasad (44a) has discussed the effect of dissipation on the mixed convective heat and mass transfer flow of a viscous fluid through a porous medium in a vertical channel bounded by flat walls. Vijayabhaskar Reddy (59) has analysed the combined influence of radiation and thermo-diffusion on convective heat And mass transfer flow of a viscous fluid through a porous medium in vertical channel whose walls are maintained at non-uniform temperatures.

#### 2. FORMULATION OF THE PROBLEM

We analyse the steady motion of viscous, incompressible fluid in a vertical channel bounded by flat walls which are maintained at a non-uniform wall temperature in the presence of a constant heat source and the concentration on these walls are taken to be constant. The Boussinesq approximation is used so that the density variation will be considered only in the buoyancy force. The viscous is taken into account in the energy equation. Also the kinematic viscosity v, the thermal conducting k are treated as constants. We choose a rectangular Cartesian system 0(x, y) with x-axis in the vertical direction and y-axis normal to the walls. The walls of the channel are at  $y = \pm L$ . The equations governing the steady flow, heat and mass transfer are

#### **Equation of continuity:**

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

**Equation of linear momentum:** 

$$\rho_e\left(u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y}\right) = -\frac{\partial p}{\partial x} + \mu\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right) - \rho g$$
<sup>(2)</sup>

$$\rho_e\left(u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y}\right) = -\frac{\partial p}{\partial y} + \mu\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right)$$
(3)

**Equation of Energy:** 

$$\rho_e C_p \left( u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) = \lambda \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) + Q + \mu \left( \left( \frac{\partial u}{\partial y} \right)^2 + \left( \frac{\partial v}{\partial x} \right)^2 \right) - \frac{\partial (q_R)}{\partial y}$$
(4)

#### **Equation of Diffusion:**

$$\left(u\frac{\partial C}{\partial x} + v\frac{\partial C}{\partial y}\right) = D_1\left(\frac{\partial^2 C}{\partial x^2} + \frac{\partial^2 C}{\partial y^2}\right) - k_1'C + k_{11}\left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2}\right)$$
(5)

#### **Equation of State:**

$$\rho - \rho_e = -\beta \rho_e (T - T_e) - \beta^{\bullet} \rho_e (C - C_e)$$
(6)

where  $\rho_e$  is the density of the fluid in the equilibrium state, Te, Ce are the temperature and Concentration in the equilibrium state, (u, v) are the velocity components along O(x, y) directions, p is the pressure, T, C are the temperature and Concentration in the flow region, p is the density of the fluid,  $\mu$  is the constant coefficient of viscosity, Cp is the specific heat at constant pressure,  $\lambda$  is the coefficient of thermal conductivity, k is the *is the magnetic* permeability of the porous medium,  $\beta$  is the coefficient of thermal expansion,  $\beta^{\bullet}$  is the coefficient of expansion with mass fraction, D<sub>1</sub> is the molecular diffusivity ,Q is the strength of the constant internal heat source, q<sub>r</sub> is the radiative heat flux and k11 is the cross diffusivity and k<sub>11</sub> is chemical reaction coefficient.

Invoking Rosseland approximation for radiation

$$q_r = -\frac{4\sigma^{\bullet}}{3\beta_R} \frac{\partial (T'^4)}{\partial y}$$

Expanding  $T'^4$  in Taylor's series about Te neglecting higher order terms.  $T'^4 \cong 4T_e^3T' - 3T_e^4$ where  $\sigma^{\bullet}$  is the Stefan Polymoun constant  $\beta_{e}$  is the Extinction coefficient

where  $\sigma^{\bullet}$  is the Stefan-Boltzmann constant  $\beta_{R}$  is the Extinction coefficient.



**Configuration of the Problem** 

(1)

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In the equilibrium state

$$0 = -\frac{\partial p_e}{\partial x} - \rho_e g \tag{7}$$

Where  $p = p_e + p_D$ ,  $p_D$  being the hydrodynamic pressure.

The flow is maintained by a constant volume flux for which a characteristic velocity is defined as

$$q = \frac{1}{L} \int_{-L}^{L} u \, d \, y \,. \tag{8}$$

The boundary conditions for the velocity and temperature fields are u = 0, v = 0 on  $y = \pm L$ 

$$T - T_{e} = \gamma(\delta x / L)$$
 on y = ±L

 $C = C_1$  on y = -L

$$C = C_2 \text{ on } y = +L \tag{9}$$

 $\gamma$  is chosen to be twice differentiable function,  $\delta$  is a small parameter characterizing the slope of the temperature variation on the boundary.

In view of the continuity equation we define the stream function  $\psi$  as  $u = -\psi_y, \, v = \psi_x$ 

The equation governing the flow in terms of  $\psi$  are

$$\left[\frac{\partial\psi}{\partial x}\frac{\partial(\nabla^2\psi)}{\partial y} - \frac{\partial\psi}{\partial y}\frac{\partial(\nabla^2\psi)}{\partial x}\right] = \nu \left(\frac{\partial^2\psi}{\partial x^2} + \frac{\partial^2\psi}{\partial y^2}\right)^2 - \beta g \frac{\partial T}{\partial y} - \beta^{\bullet} g \frac{\partial C}{\partial y}$$
(10)

$$\rho_e C_p \left( -\frac{\partial \psi}{\partial y} \frac{\partial \theta}{\partial x} + \frac{\partial \psi}{\partial x} \frac{\partial \theta}{\partial y} \right) = \lambda \nabla^2 \theta + Q + \mu \left( \left( \frac{\partial^2 \psi}{\partial y^2} \right)^2 + \left( \frac{\partial^2 \psi}{\partial x^2} \right)^2 \right) + \frac{4}{3N_1} \frac{\partial^2 \theta}{\partial y^2}$$
(11)

$$\left(-\frac{\partial\psi}{\partial y}\frac{\partial C}{\partial x} + \frac{\partial\psi}{\partial x}\frac{\partial C}{\partial y}\right) = D_{1}\left(\frac{\partial^{2}C}{\partial x^{2}} + \frac{\partial^{2}C}{\partial y^{2}}\right) + k_{11}\left(\frac{\partial^{2}T}{\partial x^{2}} + \frac{\partial^{2}T}{\partial y^{2}}\right) - k_{1}'C$$
(12)

Introducing the non-dimensional variables in (9)- (12) as

$$(x', y') = (x, y)/L, \ (\psi' = \psi/qL, \theta = \frac{T - T_e}{\Delta T_e}, C' = \frac{C - C_1}{C_2 - C_1}$$
$$p' = \frac{p_D}{\rho_e q^2}, \ \gamma' = \frac{\gamma}{\Delta T_e}$$
(13)

(Under the equilibrium state  $\Delta T_e = T_e(L) - T_e(-L) = \left(\frac{QL^2}{\lambda}\right)$ 

the governing equations in the non-dimensional form ( after dropping the dashes ) are

$$R\frac{\partial(\psi,\nabla^2\psi)}{\partial(x,y)} = \nabla^4\psi - \frac{G}{R}(\theta_y + NC_y)$$
(14)

and the energy and diffusion equations in the non-dimensional form are

$$P_{1}R\left(-\frac{\partial\psi}{\partial y}\frac{\partial\theta}{\partial x}+\frac{\partial\psi}{\partial x}\frac{\partial\theta}{\partial y}\right) = \nabla^{2}\theta + \alpha_{1} + \left(\frac{P_{1}R^{2}E_{c}}{G}\right)\left(\left(\frac{\partial^{2}\psi}{\partial y^{2}}\right)^{2} + \left(\frac{\partial^{2}\psi}{\partial x^{2}}\right)^{2}\right)$$
(15)

$$RSc\left(-\frac{\partial\psi}{\partial y}\frac{\partial C}{\partial x}+\frac{\partial\psi}{\partial x}\frac{\partial C}{\partial y}\right) = \left(\frac{\partial^2 C}{\partial x^2}+\frac{\partial^2 C}{\partial y^2}\right) + \frac{ScS_o}{N}\left(\frac{\partial^2 \theta}{\partial x^2}+\frac{\partial^2 \theta}{\partial y^2}\right) - k_1C$$
(16)

where

$$R = \frac{qL}{v} \quad (\text{Reynolds number}), \qquad G = \frac{\beta g \Delta T_e L^3}{v^2} \quad (\text{Grashof number}) \\ P = \frac{\mu c_p}{k_1} \quad (\text{Prandtl number}), \qquad E_c = \frac{\beta g L^3}{C_p} \quad (\text{Eckert number}) \\ N = \frac{\beta^{\bullet} \Delta C}{\beta \Delta T} \quad (\text{Buoyancy Number}), \qquad Sc = \frac{v}{D_1} \quad (\text{Schmidt number}) \\ \alpha = \frac{QL^2}{\Delta T C_p} \quad (\text{Heat source parameter}), \qquad S_0 = \frac{k_{11}\beta^{\bullet}}{\beta v} \quad (\text{Soret parameter}) \\ k_1 = \frac{k_1 L^2}{D_1} \quad (\text{Chemical reaction parameter}), \qquad N_1 = \frac{\beta_R \lambda}{4\sigma_{\bullet} T_e^3} \quad (\text{Radiation parameter}) \\ N_2 = \frac{3N_1}{3N_1 + 4}, \quad P_1 = PN_2, \alpha_1 = \alpha N_2 \end{cases}$$

The corresponding boundary conditions are

$$\psi(+1) - \psi(-1) = -1$$

$$\partial \psi = \partial \psi$$

$$\frac{\partial \varphi}{\partial x} = 0, \quad \frac{\partial \varphi}{\partial y} = 0 \quad at \ y = \pm 1$$
 (17a)

$$\theta(x, y) = \gamma(\delta x) \text{ on } y = \pm 1$$
 (17b)

$$C = 0$$
 on  $y = -1$ 

$$C = 1 \text{ on } y = 1$$
 (17c)

$$\frac{\partial \theta}{\partial y} = 0, \frac{\partial C}{\partial y} = 0 \quad at \quad y = 0 \tag{18}$$

The value of  $\psi$  on the boundary assumes the constant volumetric flow in consistent with the hyphothesis (8). Also the wall temperature varies in the axial direction in accordance with the prescribed arbitrary function  $\gamma(x)$ .

# **3. ANALYSIS OF THE FLOW**

The main aim of the analysis is to discuss the perturbations created over a combined free and forced convection flow due to non-uniform slowly varying temperature imposed on the boundaries. We introduce the transformation  $\overline{x} = \delta x$ 

With this transformation the equations (14) - (16) reduce to

$$R\delta \frac{\partial(\psi, F^2\psi)}{\partial(x, y)} = F^4\psi - \frac{G}{R}(\theta_y + NC_y)$$
<sup>(19)</sup>

and the energy &diffusion equations in the non-dimensional form are

$$P_{1}R\delta\left(-\frac{\partial\psi}{\partial y}\frac{\partial\theta}{\partial x}+\frac{\partial\psi}{\partial x}\frac{\partial\theta}{\partial y}\right)=F^{2}\theta+N_{2}+\left(\frac{P_{1}R^{2}E_{c}}{G}\right)\left(\left(\frac{\partial^{2}\psi}{\partial y^{2}}\right)^{2}+\delta^{2}\left(\frac{\partial^{2}\psi}{\partial x^{2}}\right)^{2}\right)$$
(20)

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$$\delta RSc \left( -\frac{\partial \psi}{\partial y} \frac{\partial C}{\partial x} + \frac{\partial \psi}{\partial x} \frac{\partial C}{\partial y} \right) = F^2 C + \frac{ScSo}{N} F^2 \theta - k_1 C$$
<sup>(21)</sup>

where  $F^2 = \delta^2 \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2}$  for small values of the slope  $\delta$ , the flow develops slowly with axial gradient of order  $\delta$ 

and hence we take

$$\frac{\partial}{\partial \overline{x}} \approx O(1)$$

We follow the perturbation scheme and analyse through first order as a regular perturbation problem at finite values of R,G,P, Sc and D<sup>-1</sup> Introducing the perturbation expansions as  $\psi(x, y) = \psi_0(x, y) + \delta \psi_1(x, y) + \delta^2 \psi_2(x, y) + \dots$ 

$$\theta (\mathbf{x}, \mathbf{y}) = \theta_0 (\mathbf{x}, \mathbf{y}) + \delta \theta_1 (\mathbf{x}, \mathbf{y}) + \delta^2 \theta_2 (\mathbf{x}, \mathbf{y}) + \dots$$

$$C (\mathbf{x}, \mathbf{y}) = C_0 (\mathbf{x}, \mathbf{y}) + \delta C_1 (\mathbf{x}, \mathbf{y}) + \delta^2 C_2 (\mathbf{x}, \mathbf{y}) + \dots$$
(22)

On substituting (22) in (19)–(21) and separating the like powers of  $\delta$  the equations and respective conditions to the zeroth order are

$$\psi_{0, yyyyy} = \frac{G}{R} (\theta_{0,y} + NC_{0,y})$$
<sup>(23)</sup>

$$\theta_{0,yy} = -\alpha_1 - \frac{P_1 R^2 E c}{G} \psi_{0,yy}^2$$
(24)

$$C_{0,yy} - k C_0 = -\frac{ScSo}{N} \theta_{0,yy}$$
(25)

With

$$\Psi_0(+1)-\Psi(-1) = -1,$$
(25.)

$$\psi_{0, y} = 0, \psi_{0, x} = 0$$
 at  $y = \pm 1$  (25a)

$$\theta o(\pm 1) = \gamma(x)$$
 at  $y = \pm 1$ 

$$C_0(-1) = 0, \ C_0(+1) = 1$$
 (25b)

and to the first order are

$$\psi_{1, yyyy} = -\frac{G}{R}(\theta_{1,y} + NC_{1,y}) + R(\psi_{0, y}\psi_{0, xyy} - \psi_{0, x}\psi_{0, yyy})$$
(26)

$$\theta_{1,yy} = P_1 R(\psi_{0,x} \theta_{0,y} - \psi_{0,y} \theta_{0,x}) - \frac{P_1 E c}{G} (R^2 \psi_{1,yy}^2)$$
(27)

$$C_{1, yy} - k C_{1} = R Sc (\psi_{0, y} C_{0, x} - \psi_{0, x} C_{0, y}) - \frac{ScSo}{N} \theta_{1, yy}$$

$$\psi_{1(+1)} - \psi_{1(-1)} = 0$$
(28)

$$\psi_{1, y} = 0, \psi_{1, x} = 0 \text{ at } y = \pm 1$$
 (29)

$$\theta_1(\pm 1) = 0 \text{ at } y = \pm 1$$
 (30)

$$C_1(-1) = 0, \ C_1(+1) = 0$$
 (31)

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Assuming Ec<<1 to be small we take the solutions as  $\psi_0(x, y) = \psi_{00}(x, y) + Ec \psi_{01}(x, y) + \dots$ 

$$\psi_{1}(x, y) = \psi_{10}(x, y) + Ec \psi_{11}(x, y) + \dots$$
  

$$\theta_{0}(x, y) = \theta_{00}(x, y) + Ec \theta_{01}(x, y) + \dots$$
  

$$\theta_{1}(x, y) = \theta_{10}(x, y) + Ec \theta_{11}(x, y) + \dots$$
  

$$C_{0}(x, y) = C_{00}(x, y) + Ec C_{01}(x, y) + \dots$$
  

$$C_{1}(x, y) = C_{10}(x, y) + Ec C_{11}(x, y) + \dots$$
  
(32)

Substituting the expansions (31) in equations (23)-(30) and separating the like powers of Ec we get the following equations

$$\theta_{00,yy} = -\alpha_1, \ \theta_{00}(\pm 1) = \gamma(\bar{x})$$
(33)

$$C_{00,yy} = 0, \quad C_{00}(-1) = 0, C_{00}(+1) = 1$$
 (34)

$$\begin{cases} \psi_{00,yyyy} = -\frac{G}{R} (\theta_{00,y} + NC_{00,y}), \\ \psi_{00}(+1) - \psi_{00}(-1) = 1, \psi_{00,y} = 0, \psi_{00,x} = 0 \text{ at } y = \pm 1 \end{cases}$$
(35)

$$\theta_{01,yy} = -\frac{P_1 R^2}{G} \psi_{00,yy}^2, \quad \theta_{01}(\pm 1) = 0$$
(36)

$$C_{01,yy} - k C_{00} = -\frac{ScSo}{N} \theta_{01,yy}, \quad C_{01}(\pm 1) = 0$$
(37)

$$\begin{cases} \psi_{01,yyyy} = -\frac{G}{R} (\theta_{01,y} + NC_{01,y}), \\ \psi_{01}(\pm 1) - \psi_{01}(-1) = 0, \\ \psi_{01,y} = 0, \psi_{01,x} = 0 \quad at \quad y = \pm 1 \end{cases}$$
(38)

$$\theta_{10,yy} = RP_1(\psi_{00,y}\theta_{00,x} - \psi_{00,x}\theta_{00,y}) \quad \theta_{10}(\pm 1) = 0$$
(39)

$$C_{10,yy} = RP_1(\psi_{00,y}C_{00,x} - \psi_{00,x}C_{00,y}) - \frac{ScSo}{N}\theta_{10,yy} \quad C_{10}(\pm 1) = 0$$
(40)

$$\begin{cases} \psi_{10,yyyy} = -\frac{G}{R} (\theta_{10,y} + NC_{10,y}) + R(\psi_{00,y}\psi_{00,xyy} - \psi_{00,x}\psi_{00,yyy}), \\ \psi_{10}(+1) - \psi_{10}(-1) = 0, \psi_{10,y} = 0, \psi_{10,x} = 0 \text{ at } y = \pm 1 \end{cases}$$
(41)

$$\theta_{11,yy} = RP_1(\psi_{00,y}\theta_{1,x} - \psi_{1,x}\theta_{00,y}), \quad \theta_1(\pm 1) = 0$$
(42)

$$C_{11,yy} = RSc(\psi_{00,y}C_{00,x} - \psi_{1,x}C_{00,y}) - \frac{ScSo}{N}\theta_{11,yy}, \quad C_{11}(\pm 1) = 0$$
(43)

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$$\begin{cases} \psi_{11,yyyy} = -\frac{G}{R} (\theta_{11,y} + NC_{11,y}) + R(\psi_{00,y}\psi_{11,xyy} - \psi_{00,x}\psi_{01,yyy} + \psi_{01,y}\psi_{00,xyy} - \psi_{01,x}\psi_{00,yyy}), \\ \psi_{11}(+1) - \psi_{11}(-1) = 0, \\ \psi_{11,y} = 0, \\ \psi_{11,x} = 0 \quad at \quad y = \pm 1 \end{cases}$$

$$(44)$$

#### 4. NUSSELT NUMBER and SHERWOOD NUMBER

The local rate of heat transfer coefficient (Nusselt number Nu) on the walls has been calculated using the formula

$$Nu = \frac{1}{\theta_m - \theta_w} \left( \frac{\partial \theta}{\partial y} \right)_{y=\pm 1}, \qquad \qquad \theta_m = 0.5 \int_{-1}^1 \theta \, dy$$

and the corresponding expressions are

$$(Nu)_{y=+1} = \frac{(-0.5\alpha_1 + Ec\,d_{25} + \delta\,d_{27})}{(\theta_m - \gamma(x))}, \quad (Nu)_{y=-1} = \frac{(-0.5\alpha_1 + Ec\,d_{26} + \delta\,d_{27})}{(\theta_m - \gamma(x))}$$
$$\theta_m = d_{29} + Ec\,d_{30} + \delta\,d_{31}$$

The local rate of mass transfer coefficient( Sherwood Number Sh) on the walls has been calculated using the formula

$$Sh = \frac{1}{C_m - C_w} \left(\frac{\partial C}{\partial y}\right)_{y=\pm 1} \text{ where } C_m = 0.5 \int_{-1}^{1} C \, dy$$

and the corresponding expressions are

$$(Sh)_{y=+1} = \frac{(d_{32} + Ec \, d_{34} + \delta \, d_{36})}{(C_m - 1)}, \quad (Sh)_{y=-1} = \frac{(d_{33} + Ec \, d_{35} + \delta \, d_{39})}{(C_m)}$$

where  $C_m = d_{38} + Ec d_{39} + \delta d_{40}$ 

where  $d_1.d_2$ ,...., $d_{40}$  are constants.

# 5. DISCUSSION OF THE NUMERICAL RESULTS

In this analysis we discuss the effect of thermo diffusion and dissipation on convective heat and mass transfer flow of a viscous chemically reacting fluid in a non-uniformly heated vertical channel in the presence of heat sources. The analysis has been carried out by employing a perturbation technique with the slope  $\delta$  of the boundary temperature as a perturbation parameter.

The axial velocity (u) is shown in figures 1 - 4 for different values of  $S_0$ ,  $\gamma$ ,  $\alpha_1$  and Ec. Fig (1) represents u with Soret parameter  $S_0$ . It is found that |u| enhances with increase in  $S_0 > 0$  and depreciates in  $|S_0|$  (<0). The variation of u with chemical reaction parameter  $\gamma$  is shown in fig (2). |u| enhances in the degenerating chemical reaction case. Fig (3) represents u with amplitude  $\alpha_1$  of the boundary temperature. It is found that higher the amplitude  $\alpha_1$ , smaller |u| in the flow region. The effect of dissipation on u is shown in fig (4). It is found that higher the dissipative heat larger |u| in the flow region.

The secondary velocity (v) which is due to the non-uniform boundary temperature is exhibited in figures 5 - 8 for different parametric values. |v| enhances in the left half and reduces in the right half of the channel with increase in the Soret parameter  $|S_0|$  (Fig 5). From fig (6) we notice that the magnitude of v enhances with increase in the chemical reaction parameter  $\gamma$ . An increase in the amplitude  $\alpha_1$  of the boundary temperature depreciates |v| in the left half and enhances in the right half of the channel (fig 7). An increase in Ec leads to an enhancement in |v| thus higher the dissipative heat larger |v| in the flow region (fig 8).

The non-dimensional temperature ( $\theta$ ) is shown in figures 9–12 for different parametric values. We follow the convention that the non–dimensional temperature is positive or negative according as the actual temperature is greater / lesser than T<sub>E</sub>. The actual temperature reduces with increase in Soret parameter S<sub>0</sub> > 0 and enhances with |S<sub>0</sub>| (<0) (fig.9). An increase in the chemical reaction parameter  $\gamma$  leads to an enhancement in the actual temperature (fig 10). With respect to the amplitude  $\alpha_1$  of the boundary temperature we notice that higher the amplitude  $\alpha_1$  larger the actual temperature (fig.11). An increase in Ec results in a depreciation in the actual temperature (fig 12). Thus higher the dissipative heat smaller the actual temperature in the entire flow region.

The non-dimensional concentration (C) is shown in figures 13–16 for different parametric values. We follow the convention that the non – dimensional concentration is positive or negative according as the actual concentration is greater / lesser than C<sub>1</sub>. An increase in S<sub>0</sub> > 0 reduces the actual concentration in the left half and enhances in the right half , while for  $|S_0|$  we notice an enhancement in C in the region -0.8, -0.6 and depreciation in the remaining region (fig 13). From fig (14) we find that the actual concentration reduces in the flow region except in a narrow region adjacent to y = -1 with increase in the chemical reaction parameter  $\gamma$ . With respect to  $\alpha_1$ , we find that the actual concentration depreciates in the left half and enhances in the right half with increase in the amplitude  $\alpha_1$  of the boundary temperature (fig 15). With respect to Ec we observe an enhancement in the actual concentration in the flow region. Thus higher the dissipative heat larger the actual concentration in the entire flow region (fig 16).



<sup>1</sup>R. Mabuni<sup>\*</sup>, <sup>1</sup>U. Rajeswara Rao, and <sup>2</sup>Prof. D. R. V. Prasada Rao / Effect of Radiation, Dissipation and Thermo-Diffusion On Convective Heat and Mass Transfer Flow of a Chemically Reacting Fluid... / IJMA- 5(6), June-2014.



<sup>1</sup>R. Mabuni<sup>\*</sup>, <sup>1</sup>U. Rajeswara Rao, and <sup>2</sup>Prof. D. R. V. Prasada Rao / Effect of Radiation, Dissipation and Thermo-Diffusion On Convective Heat and Mass Transfer Flow of a Chemically Reacting Fluid... / IJMA- 5(6), June-2014.



The rate of heat transfer (Nusselt Number) at  $y = \pm 1$  is shown in tables 1 - 4 for different values of  $S_0$ ,  $\alpha_1, \gamma$ , Ec. An increase in Soret parameter  $S_0 > 0$  depreciates |Nu| at y=+1 and enhances at y = -1 while for  $|S_0|$  (<0) it enhances at y = +1 and depreciates at y = -1. An increase in the chemical reaction parameter  $\gamma$  results in an enhancement in |Nu| at both the walls. (Tables 1 & 3). From tables 2 & 4, an increase in amplitude  $\alpha_1$  of the boundary temperature results in an enhancement in |Nu| at both the walls. With respect to Ec we find that higher the dissipative heat larger |Nu| and for further higher dissipative heat smaller |Nu| at y = -1, while at y = +1 |Nu| depreciates with the higher dissipative heat.

G	Ι	II	III	IV	V	VI	VII
$10^{3}$	0.2456	0.2333	0.2337	0.2338	0.3502	0.3629	0.3656
$3 \times 10^3$	0.2465	0.2334	0.2339	0.2340	0.3501	0.3629	0.3666
-10 <sup>3</sup>	0.2346	0.2338	0.2334	0.2336	0.3503	0.3630	0.3650
$-3 \times 10^3$	0.2375	0.2339	0.2336	0.2340	0.3506	0.3636	0.3659
S <sub>0</sub>	0.5	1	-0.5	-1	0.5	0.5	0.5
γ	0.5	0.5	0.5	0.5	1.5	2.5	3.5

Nusselt number (Nu) at y = +1 (Table - 1)

Nusselt number (Nu) at y = +1 (Table - 2)

G	Ι	II	III	IV	V	VI	VII
$10^{3}$	0.2456	0.2073	0.2670	0.3118	0.1843	0.1522	0.1297
$3 \times 10^3$	0.2465	0.2076	0.2689	0.3127	0.1842	0.1522	0.1296
-10 <sup>3</sup>	0.2346	0.2077	0.2672	0.3120	0.1845	0.1524	0.1298
$-3 \times 10^3$	0.2375	0.2078	0.2673	0.3121	0.1846	0.1525	0.1299
$\alpha_1$	0.5	0.3	0.7	0.9	0.5	0.5	0.5
Ec	0.03	0.03	0.03	0.03	0.05	0.07	0.09

#### Nusselt number (Nu) at y = -1 (Table - 3)

G	Ι	II	III	IV	V	VI	VII
10 <sup>3</sup>	0.00005	0.42374	0.42245	0.42202	1.05557	1.17947	1.20103
$3 \times 10^3$	0.00007	0.42415	0.42223	0.42158	1.05564	1.17952	1.20106
$-10^{3}$	-0.00006	0.42206	0.42336	0.42380	1.05527	1.17929	1.20089
$-3 \times 10^3$	-0.00010	0.42163	0.42360	0.42424	1.05519	1.17924	1.20085
S <sub>0</sub>	0.5	1	-0.5	-1	0.5	0.5	0.5
γ	0.5	0.5	0.5	0.5	1.5	2.5	3.5

Nusselt number (Nu) at y = -1 (Table - 4) G I Π III IV V VI VII  $10^{3}$ 0.00005 0.34551 0.54674 0.77202 0.28609 0.21606 0.17357  $3 \times 10^3$ 0.00007 0.34578 0.54692 0.77220 0.28622 0.21615 0.17364  $-10^{3}$ -0.00006 0.34439 0.54602 0.77135 0.28559 0.21569 0.17328 0.34410 -3 x 10<sup>3</sup> -0.00010 0.54585 0.77119 0.28546 0.21560 0.17321 0.9 0.5 0.3 0.7 0.5 0.5 0.5  $\alpha_1$ 0.03 0.03 0.03 0.03 0.07 0.09 Ec 0.05

#### Sherwood number (Sh) at y = +1 (Table - 5)

G	Ι	II	Ш	IV	V	VI	VII
$10^{3}$	-0.01313	-0.01260	-0.004293	-0.007747	-0.3351	-1.169	-2.122
$3 \times 10^3$	-0.01312	-0.01243	-0.004296	-0.007581	-0.3347	-1.169	-2.123
$-10^{3}$	-0.01320	-0.01333	-0.004280	-0.008380	-0.3366	-1.169	-2.119
$-3 \times 10^3$	-0.01322	-0.01352	-0.004277	-0.008530	-0.3370	-1.170	-2.119
S <sub>0</sub>	0.5	1	-0.5	-1	0.5	0.5	0.5
γ	0.5	0.5	0.5	0.5	1.5	2.5	3.5

Sherwood number (Sh) at y = +1 (Table - 6)

G	Ι	II	III	IV	V	VI	VII
$10^{3}$	-0.01313	-0.01320	-0.01574	-0.01672	-0.01459	-0.01469	-0.01519
$3 \times 10^3$	-0.01312	-0.01315	-0.01573	-0.01671	-0.01457	-0.01467	-0.01527
-10 <sup>3</sup>	-0.01320	-0.01339	-0.01579	-0.01675	-0.01468	-0.01470	-0.01538
$-3 \times 10^3$	-0.01322	-0.01344	-0.01581	-0.01676	-0.01471	-0.01476	-0.01531
$\alpha_1$	0.5	0.3	0.7	0.9	0.5	0.5	0.5
Ec	0.03	0.03	0.03	0.03	0.05	0.07	0.09

Sher wood humber (Sh) at $y = 1$ (Table 7)									
G	Ι	II	III	IV	V	VI	VII		
$10^{3}$	0.006465	0.006714	0.01393	0.01145	0.1783	0.6766	1.269		
$3 \times 10^3$	0.006297	0.006233	0.01416	0.01169	0.1770	0.6741	1.266		
-10 <sup>3</sup>	0.007154	0.008723	0.01305	0.01053	0.1836	0.6867	1.281		
$-3 \times 10^3$	0.007330	0.009248	0.01284	0.01032	0.1849	0.6892	1.284		
S <sub>0</sub>	0.5	1	-0.5	-1	0.5	0.5	0.5		
γ	0.5	0.5	0.5	0.5	1.5	2.5	3.5		

Sherwood number (Sh) at y = -1 (Table - 7)

Sher (vou humber (Sh) $u y = 1$ (ruble v)									
G	Ι	Π	III	IV	V	VI	VII		
$10^{3}$	0.006465	0.006809	0.005611	0.005040	0.006230	0.006130	0.006030		
$3 \times 10^3$	0.006297	0.006498	0.005506	0.004968	0.006063	0.005964	0.005864		
$-10^{3}$	0.007154	0.008101	0.006039	0.005332	0.006912	0.006812	0.006812		
$-3 \times 10^{3}$	0.007330	0.008437	0.006148	0.005406	0.007086	0.006986	0.006986		
$\alpha_1$	0.5	0.3	0.7	0.9	0.5	0.5	0.5		
Ec	0.03	0.03	0.03	0.03	0.05	0.07	0.09		

Sherwood number (Sh) at y = -1 (Table - 8)

The rate of mass transfer (Sherwood number) at  $y = \pm 1$  is shown in tables 5–8 for different parametric values. An increase in Soret parameter  $S_0 > 0$  depreciates |Sh| at y = +1 and enhances at y = -1 and for  $|S_0|$  (<0) it enhances at y = +1 and depreciates at y = -1. The variation of chemical reaction parameter  $\gamma$  shows that the rate of mass transfer enhances in the degenerating chemical reaction case at both the walls (Tables 5 & 7). From tables 6 & 8 we find that the rate of mass transfer enhances at y = +1 and depreciates at y = -1 with increase in the strength of the heat source and in the case of heat sink, |Sh| depreciates at y=+1 and reduces at y = -1. An increase in the amplitude  $\alpha_1$  of the boundary temperature enhances |Sh| at y = +1 and reduces at y = -1. An increase in Ec, reduces |Sh| at y = -1 and enhances at y = +1 and smaller at y = -1.

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