

**EFFECT OF NON- LINEAR DENSITY TEMPERATURE VARIATION
ON MIXED CONVECTIVE HEAT AND MASS TRANSFER FLOW OF A JEFFREY'S FLUID
THROUGH A POROUS MEDIUM IN CONCENTRIC CYLINDRICAL ANNULUS**

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ABSTRACT

In this paper, we discuss the effect of non- linear density temperature variation on mixed convective heat and mass transfer flow of a Jeffrey's fluid through a porous medium in concentric cylindrical annulus in the presence of heat sources. The equations governing the flow, heat and mass transfer have been solved by employing Gauss-Seidel iteration procedure. The effect of various governing parameters on all the flow characteristics have been discussed graphically. The Shearstress , rate of heat and mass transfer on the cylinders are evaluated numerically for different variations.

Keywords: *Jeffery-Six constant fluid, Heat and mass transfer, cylindrical annulus.*

1. INTRODUCTION

The increasing costs of energy has lead technologists to examine measures which could considerably reduce the usage of the natural source energy (fossil). Thermal insulations will continue to find increased use as engineers seek to reduce cost. Heat transfer in porous thermal insulation with in vertical cylindrical annuli provides us insight into the mechanism of energy transport and enable engineers to use insulation more efficiently. In particular design engineers require relationships between heat transfer, geometry and boundary conditions which can be utilized cost – benefit analysis to determine the amount of insulation that will yield the maximum investment. Apart from this, the study of flow and heat transfer in the annular region between the concentric cylinders has applications in nuclear waste disposal research. It is known the canisters filled with radio active rays be buried in the earth so as to isolate them from human population and is of interest to determine the surface temperature of these canisters. This surface temperature strongly depends on the buoyancy driven flow sustained by the heated surface and the possible moment of ground water past it. This phenomenon make idealized to the study of convection flow in a porous medium contained in a cylindrical annulus and extensively has been made on these lines (8, 7).

Free convection flow and heat transfer in hydro magnetic case is important in nuclear and space technology (21, 31, 32, 49). Nanda and Purushothama (19) have analyzed the free convection of a thermal conducting viscous incompressible fluid induced by a traveling thermal waves on the circumference of a long vertical circular cylindrical pipe. The solutions of the velocity and temperature fields are obtained using the long wave approximations. Ganapathi and Purushotham (10) have studied the unsteady flow induced by a traveling thermal wave imposed on the circumference of a long vertical cylindrical column of a fluid in a saturated porous medium. The analysis is carried out following White head (39), Neeraja (20) has made a study of the fluid flow and heat transfer in a viscous incompressible fluid confined in an annulus bounded by two rigid cylinders. The flow is generated by periodic traveling waves imposed on the outer cylinder and the inner cylinder is maintained at constant temperature.

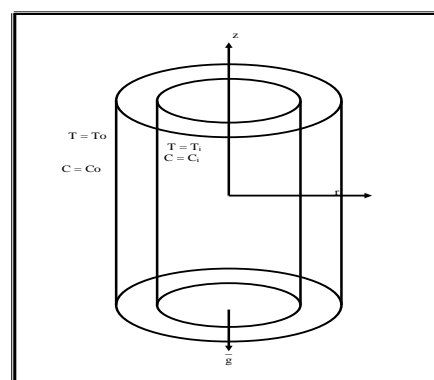
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Chen and Yuh (6) have investigated the heat and mass transfer characteristics of natural convection flow along a vertical cylinder under the combined buoyancy effects of thermal and species diffusion. The analysis is restricted to processes in which the diffusion-thermo and thermo-diffusion effects as well as the interfacial velocities from species diffusion are negligible. The surface of the cylinder is either maintained at a uniform temperature and concentration (or) subjected to uniform heat and mass flux. The conservation equations of the laminar boundary layer are solved in finite difference method. Sivanjaneya Prasad (33) has investigated the free convection flow of an incompressible, viscous fluid through a porous medium in the annulus between the porous concentric cylinders under the influence of a radial magnetic field. Antonio (5) has investigated the laminar flow, heat transfer in a vertical cylindrical duct by taking in to account both viscous dissipation and the effect of buoyancy. The limiting case of fully developed natural convection in porous annuli is solved analytically for steady and transient cases by E. Shaarawi and AL-Nimr (4). Philip (22) has obtained analytical solution for the annular porous media valid for low modified Reynolds number. Rani (28) has analyzed the unsteady convection heat and mass transfer through a cylindrical annulus with constant heat sources taking the aspect ratio ' δ ' as perturbation parameter. The governing equations have been solved by a regular perturbation method. The effect of the traveling transverse wave imposed on the boundary of the cylinders on the velocity, temperature and concentration have been discussed in detail. Srevani (36) has investigated the convective heat and mass transfer through a porous medium in a cylindrical annulus under radial magnetic field and with Soret effect. Taking G/R much less than 1 the coupled equations governing the flow, heat and mass transfer have been solved by regular perturbation method. The governing equations have been solved by regular perturbation method. Sreenivas Reddy (35) has discussed the Soret effect on mixed convective heat and mass transfer through a porous cylindrical annulus. The expressions for the velocity, temperature and concentration have been analyzed computationally for different parameters. Ramakrishna Reddy (27) has analyzed the thermo-diffusion effect on mixed convection Heat and Mass transfer through a porous medium confined in a cylindrical annulus.

A large class of real fluids does not exhibit the linear relationship between stress and the rate of strain. Because of the non-linear dependence, the analysis of the behavior of the fluid motion of the non-Newtonian fluids tend to be much more complicated and subtle in comparison with that of the Newtonian fluids. In the literature, the mechanics of non-linear fluids presents special challenges to engineers, physicists and mathematicians since the non-linearity can manifest itself in a variety of ways. One of the simplest way in which the viscoelastic fluids have been classified is the methodology given by Rivlin and Ericksen [30] and Truesdell and Noll, [37] who presents constitutive relations for the stress tensor as a function of the symmetric part of the velocity gradient and its higher (total) derivatives. In recent years there have been several studies [23,-26, 11,-14] on flows of non-Newtonian fluids, not only because of their technological significance but also in the interesting mathematical features presented by the equations governing the flow. On the other hand, it is well known that the rheological properties of many fluids are not well modelled by the Navier-Stokes equations [14]. It is not possible to obtain a single equation exhibiting all properties of all non-Newtonian fluids from available literature. That is why several models of non-Newtonian fluids are proposed. Jeffrey-six constant fluid is one of these models. Examples include various suspensions such as coal-water or coal-oil slurries, food products, inks, glues, soaps, polymer solutions, mud, blood at low shear rate, cosmetic products and many others. Al Khatib and Wilson (2) have studied the Poiseuille flow of a yield stress fluid in a channel. Flow of a visco-elastic fluid in a channel of slowly varying width was studied by Frigaard and Ryan (9). Mokhtar *et al.* (16) have studied the pulsatile MHD non-Newtonian fluid flow with heat and mass transfer through a porous medium between two permeable parallel plates. Mishra *et al.* (15) investigated a flow and heat transfer of a MHD viscoelastic fluid in a channel with stretching walls. Ali and Asghar (3) have analyzed by oscillatory channel flow for non-Newtonian fluid. Rita and Jyoti Das (29) have studied the effect of heat transfer on MHD oscillatory viscoelastic fluid flow in a channel through a porous medium. Nadeem and Akbar (17) studied the effects of temperature dependent viscosity on peristaltic flow of a Jeffrey-six constant fluid in a uniform vertical tube. Nadeem and Akbar (18) examined influence of heat and mass transfer on a peristaltic motion of a Jeffrey-six constant fluid in an annulus. Akbar *et al.* (1) discussed simulation of heat transfer on the peristaltic flow of a Jeffrey-six constant fluid in a diverging tube. Analyze the flow a Jeffrey-six constant incompressible fluid between two infinite coaxial cylinders.

The effect of heat transfer on MHD oscillatory flow of a Jeffrey fluid in a channel with slip effect at lower wall. The expressions are obtained for velocity and temperature analytically. The effects of various emerging parameters on the velocity and temperature are discussed through graphs in detail. Vasudev *et al.* (38) have discussed the effect of Heat Transfer on the peristaltic flow of a Jeffrey's Fluid through a Porous medium in a vertical annulus. Recently Sreenath *et al.* (34) have investigated the effect of quadratic density temperature variation on convection heat transfers flow of a Jeffrey's fluid in a tube and circular annulus.

In this paper, we discuss the effect of non-linear density temperature variation on mixed convective heat and mass transfer flow of a Jeffrey's fluid through a porous medium in concentric cylindrical annulus in the presence of heat sources. The equations governing the flow, heat and mass transfer have



SCHEMATIC DIAGRAM OF THE CONFIGURATION

been solved by employing Gauss-Seidel iteration procedure. The effects of various governing parameters on all the flow characteristics have been discussed graphically. The Shear stress, rate of heat and mass transfer on the cylinders are evaluated numerically for different variations.

2. FORUMLATION AND SOLUTION

We analyse the fully developed, steady laminar free convective flow of a viscous, electrically conducting Jeffrey's fluid through a porous medium confined in an annular region between two vertical co-axial porous circular pipes in the presence of heat generating sources. We choose the cylindrical polar coordinates system O (r, θ, z) with the inner and outer cylinders at r = a and r = b respectively. The fluid is subjected to the influence of a radial magnetic field (H₀ / r). Pipes being sufficiently long all the physical quantities are independent of the axial coordinate z. The fluid is chosen to be of small conductivity so that the Magnetic Reynolds number is much smaller than unity and hence the induced magnetic field is negligible compared to the applied radial field. Also the motion being rotationally symmetric the azimuthal velocity V is zero.

The boundary conditions are

$$w(a) = w(b) = 0 \tag{1}$$

$$T(a) = T_i \text{ and } T(b) = T_o \tag{2}$$

$$C(a) = C_i \text{ and } C(b) = C_o \tag{3}$$

In the hydrostatic state gives

$$-\rho_e g - p_{e,z} = 0 \tag{4}$$

Where ρ_e and p_e are the density and pressure in the static case and hence

$$-\rho g - p_z = -(\rho - \rho_e) g - p_{d,z} \tag{5}$$

Where p_d is the dynamic pressure

By using (5). We find

$$\frac{\partial p_d}{\partial r} = f(r) \tag{6}$$

Using the relation (3) – (6) the equations governing free convective heat transfer flow under no pressure gradient are

$$w_{rr} + w_r / r + (1/(1+\lambda_1))(\beta_0 g z (T - T_e) + \beta_1(T - T_e)^2 + (\beta^* g / \nu) (C - C_e) - (\nu / k) (1/(1+\lambda_1))w = 0 \tag{7}$$

$$T_{rr} + T_r / r + (Q/k_f) + (Q_1'/k_f)(C - C_e) = 0 \tag{8}$$

$$C_{rr} + C_r / r - k_1 C = 0 \tag{9}$$

Introducing the non-dimensional variables

(r', w', θ') as

$$r' = r/a, w' = w(a/\nu),$$

$$\theta = \frac{T - T_e}{T_i - T_e}, C' = \frac{C - C_e}{C_i - C_e} \tag{10}$$

The equations (7) - (9) reduce to

$$w_{rr} + (1/r) w_r - D_2^{-1} / (1+\lambda_1) w = -(G/(1+\lambda_1)) (\theta + \gamma_1 \theta^2 + NC) \tag{11}$$

$$\theta_{rr} + \theta_r / r + \alpha + Q_1 C = 0 \tag{12}$$

$$C_{rr} + C_r / r - (\gamma Sc) C = 0 \tag{13}$$

where

$$G = (\beta g a^3 (T_1 - T_e)^2 / \nu^2) \text{ (the Grashoff number)}, \quad D_2^{-1} = (a^2 / k) \text{ (the Darcy parameter)},$$

$$P = (\mu C_p / k_f) \text{ (the Prandtl number)}, \quad \alpha = \frac{QL^2}{\Delta T k_f} \text{ (the Heat Source parameter)},$$

$$Sc = \frac{\nu}{D_1} \text{ (the Schmidt number)}, \quad N = \frac{\beta^* k_{11}}{\beta \nu} \text{ (the buoyancy ratio)},$$

$$\gamma = \frac{k_1 a^2}{D_1} \quad (\text{the chemical reaction parameter}), \quad Q_1 = \frac{Q'_1 \Delta C a^2}{k_f \Delta T} \quad (\text{the radiation absorption parameter}),$$

$$s = \frac{b}{a}$$

The corresponding boundary conditions are

$$w = 0, \theta = 1, C = 1 \quad \text{on } r = 1$$

$$w = 0, \theta = 0, C = 0 \quad \text{on } r = s \tag{14}$$

The differential equations (11)-(13) have been discussed numerically by reducing the differential equations in difference equations which are solved using Gauss-Seidel Iteration method. The differential equations involving θ_0, θ_1, w_0 and w_1 are reduced to the following difference equations

$$\left(1 - \frac{h}{2r_i}\right)\theta_{i-1} - (2)\theta_i + \left(1 + \frac{h}{2r_i}\right)\theta_{i+1} + Q_1 h^2 C_i = 0 \tag{15}$$

$$\left(1 - \frac{h}{2r_i}\right)C_{i-1} - 2C_i + \left(1 + \frac{h}{2r_i}\right)C_{i+1} - (\gamma S c)C_i = 0 \tag{16}$$

$$\left(1 - \frac{h}{2r_i}\right)w_{i-1} - (2 + h^2(D_2^{-1}(1 + \lambda_1)))w_i + \left(1 + \frac{h}{2r_i}\right)w_{i+1} = -G(1 + \lambda_1)h^2(\theta_i + NC_i) \tag{17}$$

where h is the step length taken to be 0.05 together with the following conditions

$$\theta_{,0} = 1, \theta_{,21} = m_1, C_{,0} = 1, C_{,21} = m_2$$

$$w_{,0} = 0, w_{,21} = 0$$

All the above difference equations are solved using Gauss-Seidel iterative method to the fourth decimal accuracy.

3. NUSSELT NUMBER AND SHERWOOD NUMBER

The heat transfer through the pipe to the flow per unit area of the pipe surface is given by $q = k_1 \left(\frac{\partial T}{\partial r}\right)_{r=a}$ which in

the non-dimensional form is $Nu = \left(\frac{qa}{k_1(T_1 - T_e)}\right) = \left(\frac{\partial \theta}{\partial r}\right)_{r=1}$.

The mass transfer through the pipe to the flow per unit area of the pipe surface in the non-dimensional form is

$$Sh = \left(\frac{q_1 a}{D_1(C_1 - C_e)}\right) = \left(\frac{\partial C}{\partial r}\right)_{r=1}$$

4. DISCUSSION OF THE RESULTS

In this analysis we investigate the effect of chemical reaction and radiation absorption on mixed convective heat and mass transfer flow of a viscous elastically conducting fluid through a porous medium in a circular annulus between the cylinder $r = a$ and $r = b$ with non linear density temperature relation the equation governing the flow Heat and mass transfer have been solved by employing Gauss-Seidel iteration technique.

The axial velocity (w) we shown fig.1-5 for different values G,M, $D^{-1}, \alpha, N, Q_1, \lambda_1$ and γ, γ_1 Fig.1 represents w with G, M & D^{-1} , it is find that the axial velocity in the left half (1.2,1.4) to the positive and negative in the right half (1.6,1.8),it is found that the axial velocity increases in magnitude with $G > 0$,the variation of w with Hatt mann number M shows that the axial velocity deprices with increasing N-higher Lorentz force smaller w in the flow region with respect Darcy parameter D^{-1} we find depreciation in a w with the higher value of D^{-1} . Fig.2 represents the effect of heat source parameter on w. It is found that the increase in the strength of heat source enhances w in the entire flow region while increasing heat sink ($\alpha \leq 4$) enhances w everywhere except in the velocity of $r = 1.2$ where it depreciates for higher ($\alpha \geq 6$) we notice an enhancement with w everywhere in the flow region. In Fig. 3, the variation of w with buoyancy ratio N in the molecular buoyancy dominates the thermal buoyancy force, the axial velocity enhances in the buoyancy forces are in the same direction or the forces acting on the opposite direction w reduces in the left region (1.2, 1.4) and

enhances in the region(1.6, 1.8).Fig,4 represents depicts the nature of w with Q_1 , the radiation absorption parameter $Q_1(<1.5)$ it is found that w depreciates in the left half and enhances in the right half and for the higher $Q_1=1.75$ we notice an enhancement in w everywhere in the flow region and for still higher $Q_1(\geq 2.5)$, w enhances in the entire flow region.fig.5 The variation of w with chemical reaction parameter γ is shown in fig. 4. It is found that the axial velocity enhances in both degenerating and generating chemical reaction case. Fig.6 represents w with visco-elastic parameter λ_1 and density ratio γ_1 it is found that w experiences an enhancement with increasing λ_1 in the entire flow region γ_1 an increase in the density ratio γ_1 enhances w in the left half and reduces in the right half. Thus the non-linearity in the density temperature relation leads to an enhancement in the let half and depreciation in the right half.

The non-dimensional temperature (θ) is exhibited in the figs.7-15 for different parametric values. We follow the convention that the non-dimensional temperature is positive or negative according as the actual temperature (T) is greater/ lesser than T_0 , the temperature on the outer cylinder. Fig.7 represents θ with Grashof number G it is found that the actual temperature enhances with increase in G in the entire flow region. Fig.8 & 9 represents θ with M & D^{-1} . It can seen from the profiles that the higher the Lorentz force / lesser the permeability of the porous medium smaller the actual temperature. Fig.10 represents the variation of θ with heat source parameter α . We notice an enhancement in the actual temperature with increase in the strength of heat source while increasing the strength of heat sink, the actual temperature reduces in the flow region. Fig.11 represents θ with buoyancy ratio N . It is found that when the molecular buoyancy force dominates over with thermal buoyancy force the actual temperature enhances in the flow region when the buoyancy forces are in the same direction and for the forces acting in opposite directions it depreciate in the region. The effect of chemical reaction on θ is exhibited in fig.12 it can be seen from the profiles that the actual temperature enhances in the degenerating the chemical reaction cases and reduces the generating chemical reaction case. Fig.13 represents the variation of θ with radiation absorption parameter λ_1 . It is found that the actual temperature θ enhances in the left half and reduces in the right half of the flow region. Fig.14 represents θ with visco- elastic parameter λ_1 higher the values of λ_1 large the actual temperature (Fig.15), the effect of non-linear density temperature relation on θ is exhibited Fig.15. we observe an enhancement in the actual temperature with increasing density ratio γ_1 .

The Non-dimensional concentration (C) is exhibited figures 16& 17. We follow the convention that the non-dimensional concentration is positive or negative according as the actual concentration is greater or lesser than C_0 , the concentration on outer cylinder. Fig 16 represents concentrations C with Schmidt number S_c . It is found that lesser the molecular diffusivity ($S_0 \leq 1.3$) larger the actual concentration in the left half and smaller in the right half of the flow region and for further lowering of the molecular diffusivity smaller the actual concentration in the entire flow region. Fig 17 represents the effective of chemical reaction parameter γ on C . It is found that the actual concentration enhances in the degenerating chemical reaction case and in the generating chemical reaction case, the actual concentration enhances in the left half and reduces in the right half.

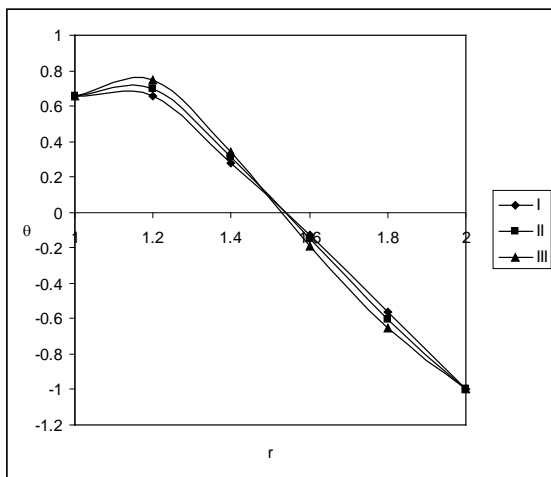


Fig. 9 : Variation of θ with D^{-1}

	I	II	III
D^{-1}	2	4	6

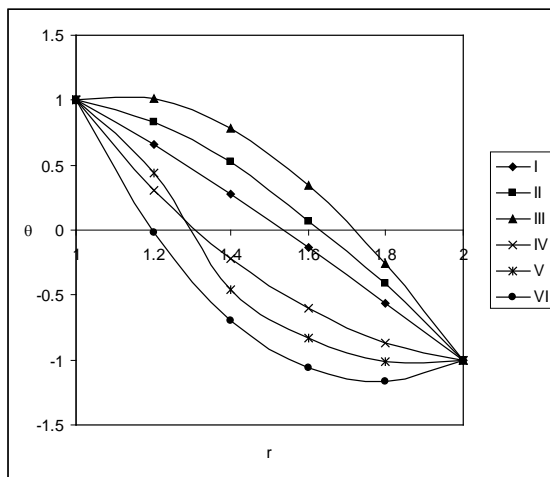


Fig. 10 : Variation of θ with α

	I	II	III	IV	V	VI
α	2	4	6	-2	-4	-6

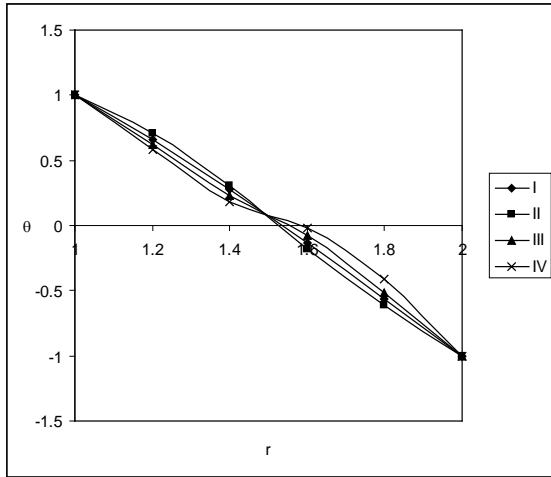


Fig. 11 : Variation of θ with N
 I II III IV
 N 1 2 -0.5 -0.8

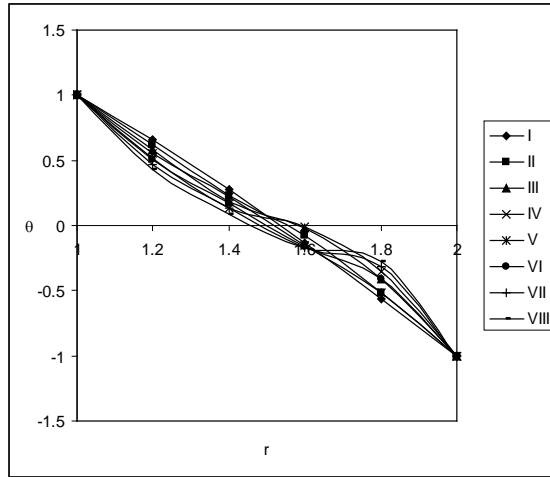


Fig. 12 : Variation of θ with k
 I II III IV V VI VII VIII
 k 0.5 1.5 2.5 3.5 -0.5 -1.5 -2.5 -3.5

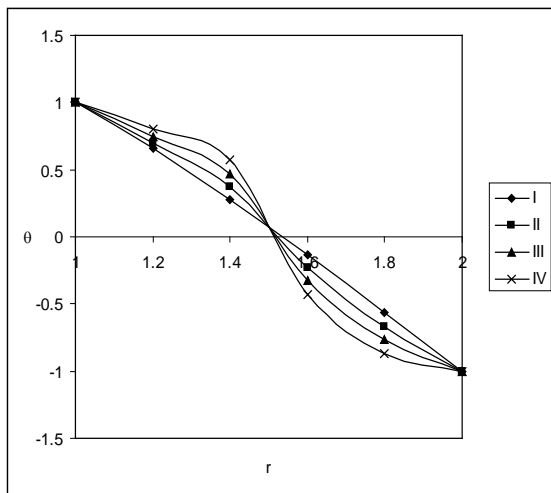


Fig. 13 : Variation of θ with Q_1
 I II III IV
 Q_1 0.5 0.65 0.75 0.95

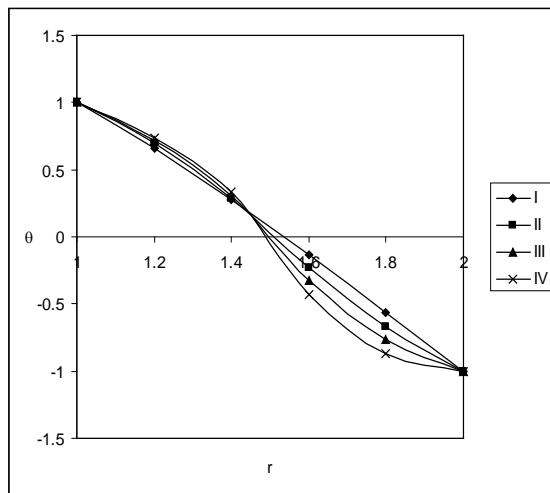


Fig. 14 : Variation of θ with λ_1
 I II III IV
 λ_1 0.1 0.3 0.5 0.7

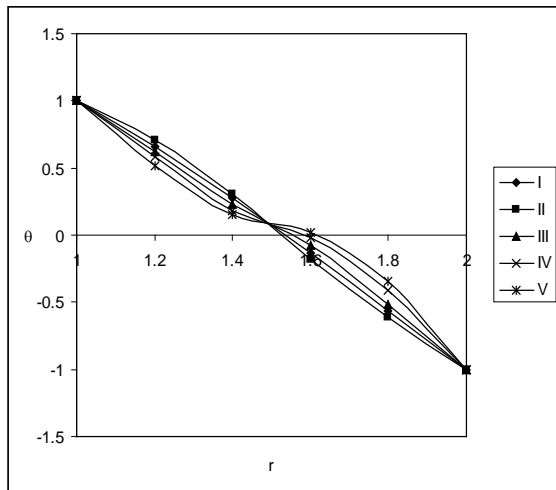


Fig. 15 : Variation of θ with γ
 I II III IV V
 γ 0.1 0.3 0.5 0.7 0.9

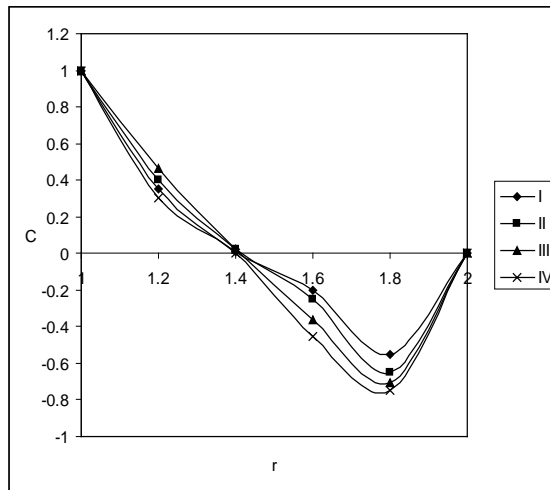


Fig. 16 : Variation of C with Sc
 I II III IV
 Sc 0.24 0.6 1.3 2.01

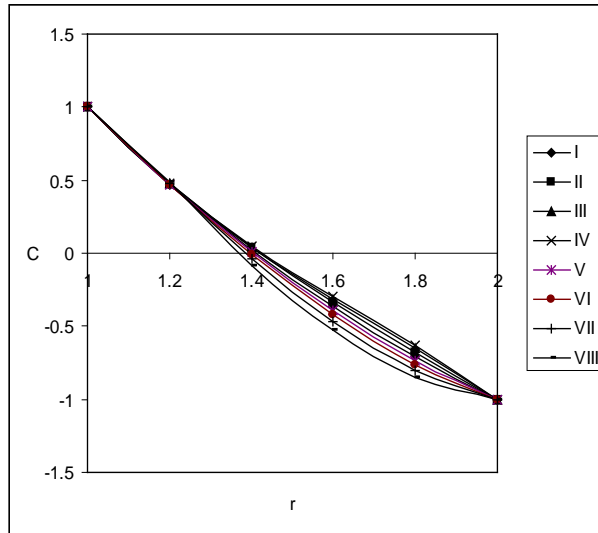


Fig. 17 : Variation of C with k

	I	II	III	IV	V	VI	VII	VIII
k	0.5	1.5	2.5	3.5	-0.5	-1.5	-2.5	-3.5

The rate of heat transfer (Nusselt Number) at the inner and outer cylinders $r = 1$ & $r = 2$ are shown in tables. 1-8 for different parametric values. We found that an increase in $\alpha \geq 0$ reduces the rate of heat transfer at $r = 1$ and enhances at $r = 2$ while it enhances with α at both the cylinders. The variation of Nu with M & D^{-1} shows that higher the Lorentz force / lesser the permeability of porous medium larger Nu at $r = 1$ and smaller at $r = 2$. Tables 1-4 with respect to the chemical reaction parameter γ we find that the rate of Heat transfer reduces at $r = 1$ and enhances at $r = 2$ in the degenerating chemical reaction case. While generating case Nu enhances at $r=1$ and reduces $r = 2$ for all $\alpha > 0$ and for $\alpha < 0$, Nu enhances at $r = 1$ and $r = 2$. An increase in the radiation absorption parameter Q_1 depreciates Nu on both cylinders for $\alpha \geq 0$ and for $\alpha < 0$, Nu reduces at $r = 1$ and enhances $r = 2$ (tables 2-5). The variation of Nu with visco-elastic parameter λ_1 shows that an increasing λ_1 reduces Nu at $r=1$ and enhances at $r=2$. An increase in the density ratio γ reduces Nu for $\alpha > 0$ and enhance for $\alpha < 0$ at $r = 1$ while at $r = 2$, Nu enhances for all α (tables 3-6).

The rate of mass transfer (Sherwood number) at $r = 1$ and $r = 2$ is shown in tables. 7&8. It is found that the rate of mass transfer enhances in magnitude with increase in Schmidt number Sc . Also the rate of mass transfer depreciates in the degenerating chemical reaction case and enhances in the generating chemical reaction case at both the cylinders (tables 7&8).

Table – 1: Nusselt Number (Nu) at $r = 1$

α	I	II	III	IV	V	VI	VII
2	-1.609	-1.654	-1.692	-1.614	-1.618	-1.505	-1.692
4	-0.51	-0.581	-0.628	-0.52	-0.527	-0.402	-0.6
-2	-3.823	-3.851	-3.885	-3.825	-3.828	-3.729	-3.889
-4	-4.901	-4.942	-4.98	-4.906	-4.191	-4.813	-4.96
M	2	5	10	2	2	2	2
D^{-1}	2	2	2	4	6	2	2
N	1	1	1	1	1	2	-0.5

Table – 2: Nusselt Number (Nu) at $r = 1$

α	I	II	III	IV	V	VI	VII	VIII	IX
2	-1.609	-1.48	-1.351	-1.223	-1.607	-1.606	-1.612	-1.617	-1.623
4	-0.51	-0.382	-0.253	-0.125	-0.507	-0.505	-0.515	-0.521	-0.53
-2	-3.823	-3.693	-3.564	-3.434	-3.823	0.0984	-3.823	-3.824	-3.826
-4	-4.901	-4.772	-4.642	-4.513	-4.9	-1.612	-4.9	-4.93	-4.999
Q_1	0.5	1.5	2.5	3.5	0.5	0.5	0.5	0.5	0.5
γ	0.5	0.5	0.5	0.5	1.5	2.5	-0.5	-1.5	-2.5

Table – 3: Nusselt Number (Nu) at r = 1

α	I	II	III	IV	V	VI	VII	VIII	IX
2	-1.609	-1.575	-1.539	-1.499	-1.457	-1.607	-1.606	-1.604	-1.603
4	-0.51	-0.467	-0.421	-0.372	-0.321	-0.507	-0.505	-0.505	-0.499
-2	-3.823	-3.795	-3.765	-3.732	-3.697	-3.823	-3.823	-3.823	-3.823
-4	-4.901	-4.87	-4.837	-4.8	-4.762	-4.903	-4.904	-4.905	-4.907
λ	0.1	0.3	0.5	0.7	0.9	0.1	0.1	0.1	0.1
γ_1	0.01	0.01	0.01	0.01	0.01	0.03	0.05	0.07	0.09

Table – 4: Nusselt Number (Nu) at r = 2

α	I	II	III	IV	V	VI	VII	VIII
2	-1.904	-1.886	-1.866	-1.902	-1.9	-1.971	-1.86	-1.861
4	-2.803	-2.783	-2.765	-2.801	-2.799	-2.86	-2.776	-2.779
-2	-0.107	-0.067	-0.031	-0.102	-0.098	-1.196	-0.032	-0.025
-4	0.7685	0.8347	0.7852	0.7671	0.7447	0.8689	0.8603	0.8703
M	2	5	10	2	2	2	2	2
D ⁻¹	2	2	2	4	6	2	2	2
N	1	1	1	1	1	2	-0.5	-0.8

Table – 5: Nusselt Number (Nu) at r = 2

α	I	II	III	IV	V	VI	VII	VIII	IX
2	-1.904	-1.74	-1.576	-1.413	-1.906	-1.908	-1.901	1.897	-1.893
4	-2.803	-2.639	-2.475	-2.311	-2.807	-2.81	-2.799	-2.794	-2.787
-2	-0.107	0.155	0.2174	0.3799	-0.107	-1.44	-0.107	-0.107	-0.107
-4	0.7685	0.9305	1.0923	1.2541	0.7696	-1.901	0.7671	0.7651	0.7623
Q ₁	0.5	1.5	2.5	3.5	0.5	0.5	0.5	0.5	0.5
γ	0.5	0.5	0.5	0.5	1.5	2.5	-0.5	-1.5	-2.5

Table – 6: Nusselt Number (Nu) at r = 2

α	I	II	III	IV	V	VI	VII	VIII	IX
2	-1.904	-1.921	-1.939	-1.959	-1.981	-1.904	-1.904	-1.904	-1.904
4	-2.803	-2.819	-2.837	-2.856	-2.876	-2.804	-2.805	-2.805	-2.806
-2	-0.107	-0.137	-0.17	-0.204	-0.24	-0.106	-0.104	-0.103	-0.101
-4	0.7685	0.7263	0.6814	0.6343	0.5855	0.7719	0.7752	0.7785	0.7817
λ_1	0.1	0.3	0.5	0.7	0.9	0.1	0.1	0.1	0.1
γ_1	0.01	0.01	0.01	0.01	0.01	0.03	0.05	0.07	0.09

Table – 7: Sherwood Number (Sh) at r = 1

Sc	I	II	III	IV	VI	VII
0.24	-8.168	-7.83	-7.567	-8.605	-9.179	-9.949
0.6	-16.01	-15.06	-14.28	-17.18	-18.68	-20.63
1.3	-31.04	-28.92	-27.17	-33.63	-36.89	-41.08
2.01	-46	-42.73	-40.01	-49.99	-54.98	-61.4
γ	0.5	1.5	2.5	-0.5	-1.5	-2.5

Table – 8: (Sherwood Number (Sh) at r = 2)

Sc	I	II	III	IV	VI	VII
0.24	3.051	2.5383	2.0915	3.6495	4.3637	5.2404
0.6	9.3249	8.3323	7.4835	10.505	11.943	13.741
1.3	21.584	19.65	18.013	23.908	26.765	30.376
2.01	34.109	31.207	28.759	37.611	41.932	47.417
γ	0.5	1.5	2.5	-0.5	-1.5	-2.5

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